Flexual Buckling of Structural Glass Columns. Initial Geometrical Imperfection as a Base for Monte Carlo Simulation

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Abstract. In this paper Monte Carlo simulations of structural glass columns are presented. The simulation was performed according to the analytical second order theory of compressed elastic rods. A previous research on shape and size of initial geometrical imperfections is briefly summarized. An experimental analysis of glass columns that were performed for evaluation of equivalent geometrical imperfections is mentioned too.

Keywords
Structural glass, flexural buckling, geometrical imperfections, Monte Carlo simulation, variation coefficient, statistical distribution, Southwell’s plot.

1. Introduction

Glass has been established as a material of load carrying members and structures in the end of twentieth century and its importance still grows today (Fig. 1), but European design code for static design of glass structures is still in progress.

Due to slender of glass members it is necessary to check them on stability problems - flexural buckling of columns or lateral torsional buckling of beams or their interaction (flexural - lateral torsional buckling) of beam - columns. Design methods of steel and timber structures are not completely usable for glass structures because of several differences (initial imperfections, brittle behaviour and laminated glass behaviour) [2]. The shape and size of initial geometrical imperfections are still poorly published in recent publications; the largest study on this problem was performed by Belis et al. [3]. Pesek and Melcher [4] followed on their work. Behaviour of imperfect columns and beams under load was published in [5]. Equation (1) describe (according to the second order theory) dependency of deformation \( f(w_0)_x \) of axially loaded imperfect column on amplitude of initial imperfection. Using equation (2) normal stresses at mid - span can be calculated. Sinusoidal shape of overall bow imperfection is considered in accordance with results of [3] and [4]. Flexural deformation increases with increasing amplitude of initial imperfection.

\[
f(w_0)_x = w_0 \frac{N}{N_{cr}} - N \cdot \frac{\cos \frac{\pi \cdot x}{L}}{\frac{1}{1 - \frac{N}{N_{cr}}}} = w_0 \left(\frac{1}{1 - \frac{N}{N_{cr}}}\right) \cdot \cos \frac{\pi \cdot x}{L}
\]

\( f(w_0)_x \) — flexural deformation of imperfect column
\( w_0 \) — initial imperfection amplitude
\( N \) — axial load
\( N_{cr} \) — critical load
\( L \) — length of column
\( x \) — distance from mid - span

Fig. 1: Flexural buckling of glass column, load-deflection curve of perfect and imperfect member.
where:

\( w_0 \) - is amplitude of initial geometrical imperfection [mm],
\( A \) - section area [mm\(^2\)],
\( W \) - section modulus to weak axis [mm\(^3\)],
\( N \) - normal compressive force [N],
\( N_{cr} \) - Euler’s critical force [N],
\( L \) - column (buckling) length [mm] and
\( x \) - point of interest, distance from mid-span [mm].

It is necessary to know size and shape of initial geometrical imperfections to make both types of simulations - analytical simulations according to the second order theory and numerical analysis according to the large deflection theory [6].

Actually there are two types of overall bow imperfection - initial geometrical imperfection (imperfection of unloaded specimen) and equivalent initial geometrical imperfection including three types of imperfections: (i) geometrical imperfections (geometrical curvature of beam or column), (ii) structural imperfections (actual point of load application etc.) and (iii) physical imperfections (residual stresses, inhomogeneity of the material). Equivalent initial geometrical imperfection is a result of Southwell’s plot in evaluation of experimental testing. In this paper the both types of geometrical imperfections are mentioned.

The geometrical imperfection of the guiding rail was deducted from measuring the initial shape imperfections of the same glass specimen twice: once in the conventional position \((u_0, u_{ncorr,1})\) and once in the mirrored position \((u_0, u_{ncorr,2})\).

Initial geometrical imperfections (global bow) were measured on 33 specimens. Specimens tested on flexural buckling are listed in Tab. 1.

1.1. Shape and Size of Initial Imperfections

Imperfect shapes of specimens tested on flexural buckling are plotted in Fig. 4 where specimens FB1 - FB3 are made of ESG 12, spec. FB4 - FB6 are made of VG 66.2, spec. FB7 - FB12 are made of VSG 66.3 and spec. FB13 - FB15 are made of VSG 444.33. The shapes are mostly symmetrical, only two of them (FB15 and FB14 - both made of laminated safety glass) are significantly asymmetrical. Results of this research were published and discussed in [4] in detail. The biggest initial overall bow imperfection was measured on specimen FB15 - 2.423 mm, which is 1.615 mm/m. The smallest initial overall bow imperfection was measured in specimen FB5 - 0.312 mm, which is 0.208 mm/m. Measured sizes of imperfections confirmed that overall bow imperfections of fully tempered glass are significantly greater than for annealed glass where imperfections are almost negligible. The shapes could be approximated using sinus function and parabola as well. Due to agreement with analytical solution an application of sinus wave is recommended.

Statistical evaluation was carried out in Statistica software [13]. To make summary statistic for all specimens is problematic due to different composition of specimens.
2. EXPERIMENTAL TESTING OF STRUCTURAL GLASS COLUMNS

Tested specimens are listed in Tab. 1.

2.1. Test Set - up

Test set - up is plotted in Fig. 5. Hinged supports were ensured by steel coulters fitted on both ends of specimen. Coulter was equipped with cutting edge which fits into the conical notch of the bearing plate. Timber pads situated between steel coulter and the glass specimen avoided direct contact of the steel and the glass which may cause a failure by local stress concentrations in contacts. The specimen was placed in a steel frame consisting of steel girders and columns. Loading force was generated by manually operated hydraulic press. Loading force, vertical deflection and horizontal (lateral) deflection at mid-span were measured using force transducer, LVDT and wire sensor respectively. Normal stresses at mid-span were measured at selected specimens using strain - gauges glued to sanded glass. Tested specimens were loaded by static force and loading rate was determined by the press cylinder pull(approximately 0.075 mm.s$^{-1}$).

2.2. Results

Force - lateral deflection curves are plotted in graphs in Fig. 6. Force - lateral deflection curves for all the specimens with the exceptions of laminated double glass with PVB foil (VG 66.2) have an increasing tendency from zero up to failure - it means that the tested specimen was some elastic material [5]. The specimen with PVB foil has curves with decreasing tendency from the point of maximal load - it is characteristic for elastic - plastic materials. Plastic behaviour is caused by PVB foil which has low shear modulus at longer load duration.

Results (ultimate load capacity of each specimen) were statistically evaluated according to the EN 1990 [9], annex D - Design assisted by testing. Calculated mean values, characteristic values and design values are plotted in graph in Fig. 7. Presented values were calculated assuming normal statistical distribution and using equations for unknown variation coefficient.
2.3. Buckling Curves Approach

The final goal of the research is to develop EC buckling curves (parameters $\alpha_{\text{imp}}$ and $\alpha_0$ that characterizes buckling curve) for simple static design and calculation of buckling resistance $N_{b,Rd}$ - see equations (3)-(5).

\[
N_{b,Rd} = \chi \cdot A \cdot f_{g,d}
\]

\[
\chi = \frac{1}{\Phi + \sqrt{\Phi^2 - \lambda^2}}
\]

\[
\Phi = 0.5 \cdot \left[ 1 + \alpha_{\text{imp}} \cdot (\overline{x} - \alpha_0) + \overline{x}^2 \right]
\]

where:

- $A$ - is cross section area [mm$^2$],
- $f_{g,d}$ - design value of glass tensile strength [MPa],
- $\chi$ - reduction factor [-] and
- $\overline{x}$ - non-dimensional slenderness [-].

In Fig. 8, buckling curve according to Amadio and Bedon [10, 11] and EC3 [12] curve care plotted. Euler’s hyperbola is plotted only for comparison. In the graph results of experiments are plotted by orange marks - reduction factors $\chi_{\text{test}}$ were calculated using equation (6) where $N_{\text{ult,test}}$ is maximal normal force measured during test and $N_{Rk}$ is cross section area $A$ multiplied by the characteristic glass strength $f_{g,k}$.

\[
\chi_{\text{test}} = \frac{N_{\text{ult,test}}}{N_{Rk}}
\]

Experimentally determined reduction factors $\chi_{\text{test}}$ are in all cases higher than reduction factors calculated according to the buckling curves - they are above these curves, in means that calculated buckling resistance is on the safe side.

2.4. Equivalent Geometrical Imperfections

It is possible to use Southwell’s plot to determine of Euler’s critical force $N_{cr}$ and equivalent geometrical initial imperfection $e_{0,ckv}$. Equivalent geometrical imperfection is imperfection that converts all imperfections (geometrical, physical and structural) into one geometrical imperfection. It is convenient because the determination of structural and physical imperfections is difficult and their subsequent application in numerical models or other simulations is problematical.

The principle of Southwell’s plot is shown in Fig. 9. Linear function (blue line) is approximated from points only in area b, because of elimination of gaps in test setup (area a) or plastic states in the end of tests (area c). Calculated equivalent imperfections were statistically evaluated, the mean value of 15 specimens is 1.368 mm (that is 0.818 mm/m) and the imperfection has lognormal distribution - see Fig. 10 on the left. But the recommendation of JCSS [15] is to use normal statistical distribution for geometrical imperfections - histogram for normal distribution of doubled set of measured imperfections is shown in Fig. 10 on the right. 5 % quantiles are 3.591 mm (2.148 mm/m) and 6.665 mm (3.986 mm/m) for normal and lognormal distribution respective. Curvature is approximately $L/250$ in the case of lognormal distribution.
3. Monte Carlo Simulations

In the frame of experimental testing low number of specimens were tested. All tested specimens had extremely high non-dimensional slenderness. From limited results it wasn’t possible to verify buckling curves reliably. Monte Carlo simulation offer to verify reliability and accuracy of developed buckling curves for all spectre of non-dimensional slenderness. It requires suitable choice of input parameters.

3.1. Input Parameters

Stochastic values of some input parameters are unavailable; in this case they were estimated. All input parameters for laminated double glass are listed in Tab. 2.

Glass strength \( f_{b,ki} \) is not a material constant because of fracture behaviour. It depends on size and number of flaws, residual stress, load history, environmental conditions etc. Statistical distribution is best fitted by two-parameter Weibull distribution [14], but these parameters are not presented. Lognormal distribution was used for simplification. Mean value and standard deviation were determined so that 5% quantile is 120 MPa - characteristic value for fully tempered glass. Mean and standard deviation of equivalent geometrical imperfection \( \delta_{0,ekv} \) were determined so that 5% quantile is 4 mm/m - that is result of Southwell’s plot of experimental testing (exactly it is 3.986 mm/m for lognormal distribution). Glass thickness \( t_1 \) and \( t_2 \) hasn’t unambiguous statistic distribution due to continual float process. The actual histogram is composed and has two or more significant peaks. Each random selection has different distribution shape. Parameters of uniform distribution corresponded with allowable deviation \( \pm 0.2 \) mm for thickness 6 mm according to [8].

Parameters of buckling length \( L_{cr} \) were determined to get wide spectre of non-dimensional slenderness. Interlayer shear modulus \( G_{int} \) has constant value 1.0 MPa.

3.2. Evaluation of the Simulation

Using MS excel 1000 Monte Carlo simulations were carried out. According to the second order theory were calculated deflections and normal stresses in glass columns with random combinations of input parameters - see equations (1) and (2). Ultimate limit state (buckling strength \( N_{ult,MC} \)) was considered when normal stress exceeds strength of glass. Then reduction factors were calculated according to the equation (6) where \( N_{ult,test} \) was replaced by \( N_{ult,MC} \). These reduction factors are plotted in graphs in Fig. [11]. In graphs buckling curves according Amadio and Bedon (red line [10, 11]), EC3 (orange line [12]) and new proposed by author (blue line) are plotted. Reduction factors are plotted by red and green colour - green colour is used when reduction factor is above the buckling curve and red colour is used when reduction factor is under buckling curve. In the case of green colour the simulation is on the safe side, in the case of red colour it is on the unsafe side. In graphs Euler’s hyperbola is plotted by grey line and blue crosses are reduction factors obtained from experimental testing.

EC3 buckling curve c is conservative for low slender columns, for slenderness from 2.0 to 5.0 there are 9 unsafe cases. Amadio and Bedon buckling curve is unsafe for very low non-dimensional slenderness, where 34 unsafe cases were obtained. Author proposes new curve with parameters listed in Tab. 3 where only 1 case is unsafe, but curve is very uneconomic for slenderness up to 2.0.

4. Conclusions

On basis of experimental testing and measuring of initial imperfections 1000 Monte Carlo simulations were carried out. New buckling curve was developed according to the results of the simulation. Further research should be target to carry out the same simulation using numerical analysis according to the large deflection theory. All simulations could be verified by additional experiments.

Acknowledgment

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References

Tab. 2: The list Monte Carlo simulation input parameters

<table>
<thead>
<tr>
<th>Variable</th>
<th>Designation</th>
<th>Statistical distribution</th>
<th>Mean</th>
<th>Standard deviation</th>
<th>Variation coefficient</th>
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<tr>
<td>Section width</td>
<td>b</td>
<td>normal</td>
<td>150 mm</td>
<td>0.75 mm</td>
<td>0.005</td>
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<tr>
<td>Interlayer thickness</td>
<td>t_{int}</td>
<td>normal</td>
<td>0.76 mm</td>
<td>0.0038 mm</td>
<td>0.005</td>
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<tr>
<td>Glass Young’s modulus</td>
<td>E</td>
<td>normal</td>
<td>70 GPa</td>
<td>3.5 GPa</td>
<td>0.05</td>
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<td>Glass tensile strength</td>
<td>f_{b,k}</td>
<td>lognormal</td>
<td>5.73 MPa</td>
<td>0.573 MPa</td>
<td>0.1</td>
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<tr>
<td>Amplitude of imperfection</td>
<td>w_{0,ekv}</td>
<td>lognormal</td>
<td>1.191 mm</td>
<td>0.119 mm</td>
<td>0.1</td>
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<td>Buckling length</td>
<td>L_{cr}</td>
<td>uniform</td>
<td>30 mm</td>
<td>1600 mm</td>
<td></td>
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<td>Glass thickness</td>
<td>t_1; t_2</td>
<td>uniform</td>
<td>5.8 mm</td>
<td>6.2 mm</td>
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Tab. 3: Summary of Monte Carlo simulations

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<th>Buckling curve</th>
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<th>α_{imp}</th>
<th>Number of unsuitable simulations</th>
<th>P_f</th>
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<td>EC3 c</td>
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<td>9</td>
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<td>author c</td>
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<td>0.65</td>
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